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IN THE SPECIFICATION:

Please amend the paragraphs on page 5, lines 16-28 as follows:

The rotor 4 is generally disc shaped. A shaft 2 extends through the centre of the disc, protruding on both sides in a direction that is normal to the general plane of the disc, so that it is coaxial with the rotor 4. On one face of the disc a plurality of impeller vanes 1 is arranged in an annular array. The impeller vanes have a curved profile shaped in accordance with the performance requirements of the blower. The impeller is moulded over a magnetisable element ring 3. The element ring 3 is subsequently magnetised to become the permanent magnetic poles of the rotor. In use the poles interact with a commutated stator to provide a torque to the rotor/impeller assembly.

Between the inner surface of the magnetisable element ring 3 and the outer surface of the shaft 2 is a hub portion 14. The inner portion of the hub 14 extends upward along the outer surface of the shaft 2 forming a shoulder portion 17. The upperside of the hub 14 is a thin shell. The underside of the hub portion 14 is a plurality of annularly spaced ribs 15. Shaft 2 may include a groove 5 adapted to receive a circlip for the purpose of securing the rotor.

Please amend the paragraph on page 5, line 29 through page 6, line 8 as follows:

Performance of the encapsulated rotor will be detrimentally effected by any imbalance arising from asymmetry of the components about its axis of rotation. In traditional designs where the impeller is not integrally formed to include the permanent magnets of the rotor, the impeller will be comparatively lightweight. Consequently there is less potential for significant imbalance when the impeller is comparatively lighter. In an encapsulated rotor such as that described in the

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present invention the integral forming of the permanent magnets with the impeller blades dramatically increases the potential for significant significant, rotational imbalance.

Consequently it is advantageous for rotors formed according to the present invention to be as close as possible to being balanced once the overmoulding process has been completed. In order to achieve this the present invention precisely locates and maintains the position of the shaft and magnet(s) in the overmoulding process.

Please amend the paragraph on page 6, line 27 through page 7, line 5 as follows:

In order to align the magnet magnetisable ring 3 coaxially with the mould cavity a number of alignment pins 13 are provided which extend into the mould cavity around the periphery and substantially perpendicular to the rods 12 as shown in Figure 8. Alignment pins 13 are extendable and retractable in order to facilitate alignment of the magnetisable ring 3 and the extraction of the completed rotor product. In the preferred embodiment of the present invention there are three alignment pins 13 arranged in the horizontal plane, 120 degrees apart. The alignment pins contact the outer surface of ring 3, in order to provide coaxial alignment between the mould cavity and ring. The alignment pins 13 also allow molten plastic to flow around them encapsulating the rotor. In a similar manner to detents 6, the alignment rods form outer detents apertures 7 in the finished rotor product.

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Please amend the paragraph on page 10, lines 13-17 as follows:

The wall thickness of the material encapsulating the magnetisable disc ring 3 is increased around the circumference 34 in order to provide sufficient material which can be removed during the balancing process. Similarly the top surface of the impeller cap 20 is provided with a raised annular ring 16 of plastic material which can be later removed for the purposes of balancing the rotor.